

Skin Friction and Heat Transfer Increase due to Free-Stream Turbulence



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**NORDITA Spring Programme on Turbulent
Boundary Layer, April 29-30, 2010, Stockholm,
Sweden**

Outline

- Introduction
 - Motivation
 - Review
- Numerical Simulation
 - Spatial Boundary Layer
 - LES Modelling
 - Free-Stream Turbulence Generation
 - Main Parameters
- Results
 - Validation of LES without FST
 - Results
- Conclusions & Outlook



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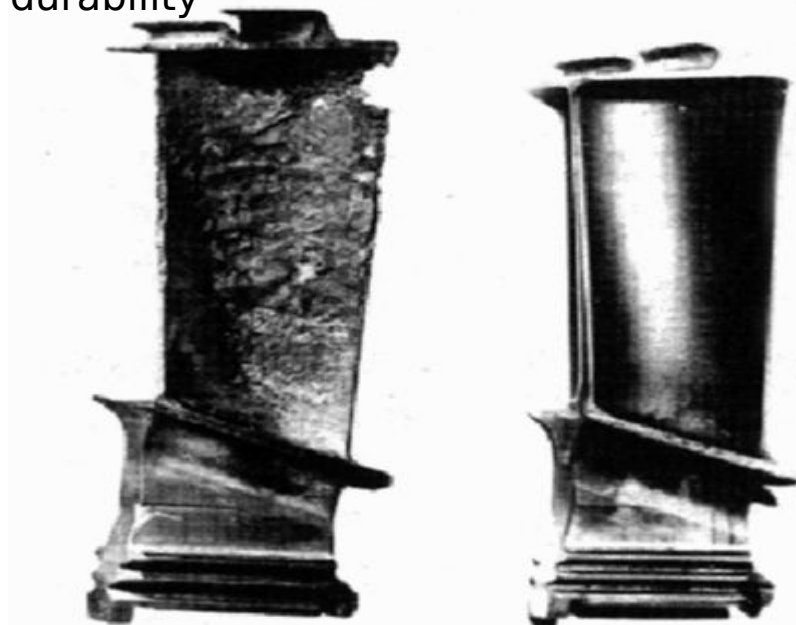
Motivation

Basic understanding of wall-bounded flows with heat transfer, especially flows with turbulent free stream:

- **Crucial in industrial situations:**
 - e.g. turbomachines, reactors, gas turbine, combustion chambers of motors etc.
- accurate prediction of both boundary layer development and convective heat transfer → cycle efficiency, cooling design, hardware durability



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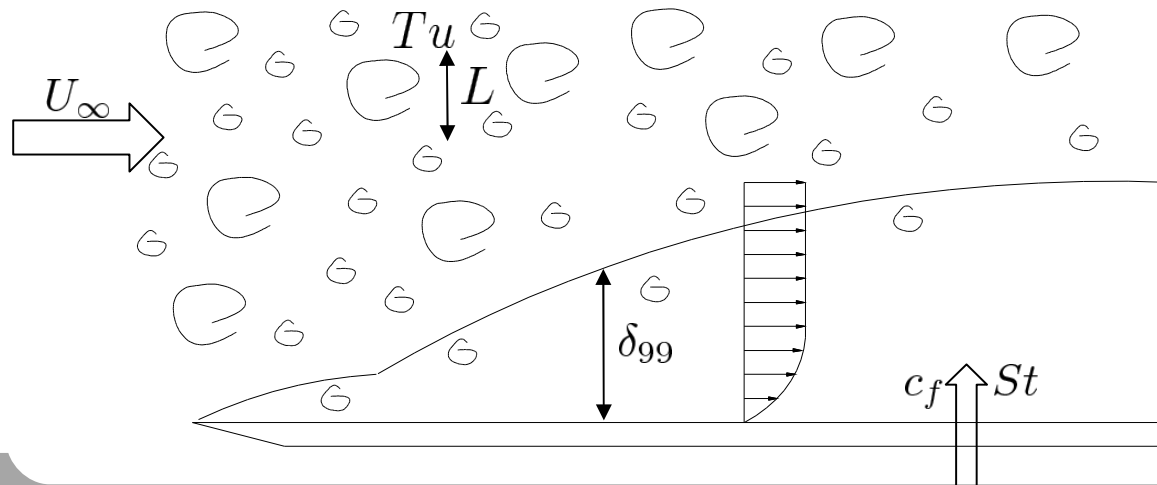


A destroyed turbine blade due to hot corrosion, *Eskner, (2004)*

Motivation

Basic understanding of wall-bounded flows with heat transfer, especially flows with turbulent free stream:

- **Crucial in academic situations:**
 - Understanding of the **heat transfer subjected to FST** is limited (e.g. Reynolds Analogy)
 - It is difficult when measuring close to the wall \rightarrow numerical simulations are needed
 - Effects of the **large-scale structures** on the heat transfer on the wall are not well-understood
 - Clearly a need for well resolved numerical simulations of heat transfer in TBL with ambient FST
(Jacobs & Durbin, 2000, Péneau *et al.*, 2000)



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Previous Studies

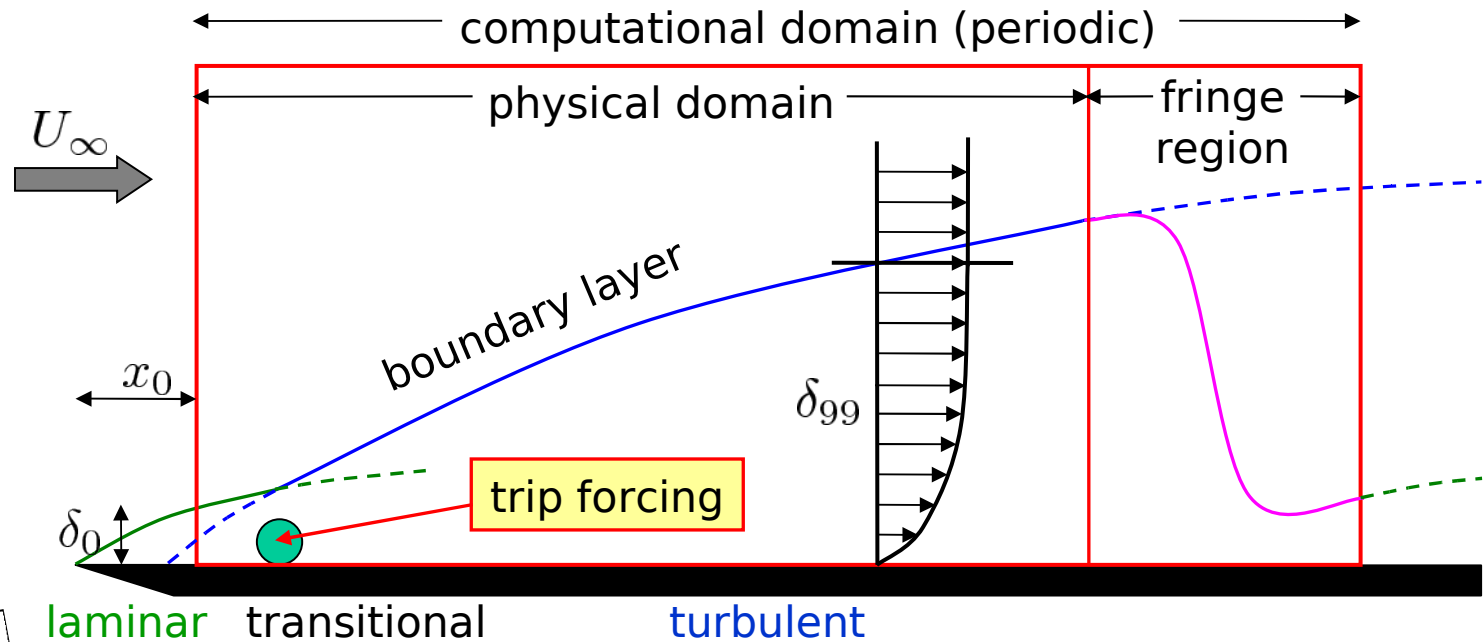
		Re_θ	Tu %	c_f %	St %
Simonich & Bradshaw, 1978	Exp	6000	4.7	10	30
Blair, 1983	Exp	6000	3.7	15	20
Péneau <i>et al.</i> , 2000	LES	1000	21	10	20
Jacobs & Durbin, 2000	DNS	800	1	5	12



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- c_f and St increase with Tu , St increase at a higher rate than c_f → Reynolds Analogy Factor is increasing with Tu
- The relative increase of c_f and St depend on Tu , L , Re
- Wake region is depressed even vanished for high Tu
- How much increase is different from one to another
- No real explanations provided

Spatial Boundary Layer



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- Fully spectral method: Fourier/Chebyshev tau method
- Periodic b.c. in the wall-parallel directions, no-slip at lower wall, Neumann conditions at upper boundary. Air flow, $Pr=0.71$
- Fringe region (volume force) to enforce laminar Blasius inflow
- Trip forcing to induce "natural" laminar-turbulent transition
- Code **SIMSON** (Chevalier *et al.* 2007) on 1024 processors BG/L

Relaxation Term (RT) Model

Recently RT model for heat transfer is implemented

$$\frac{\partial \tau_{ij}}{\partial x_i} = \chi H_N * \tilde{u}_i$$

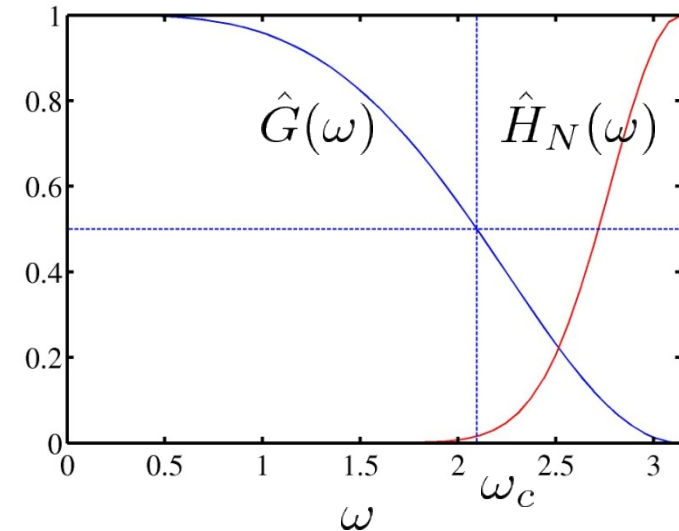
$$\frac{\partial \sigma_i}{\partial x_i} = \chi_\theta H_N * \tilde{\theta} \quad \chi_\theta = \frac{\chi}{Pr_t}$$

- High-pass filter: $\hat{H}_N = (1 - \hat{G})^{N+1}$
with $N = 5$ and $\omega_c = 2\pi/3$

Model Coefficient $\chi = 0.2$ with $Pr_t = 0.6$

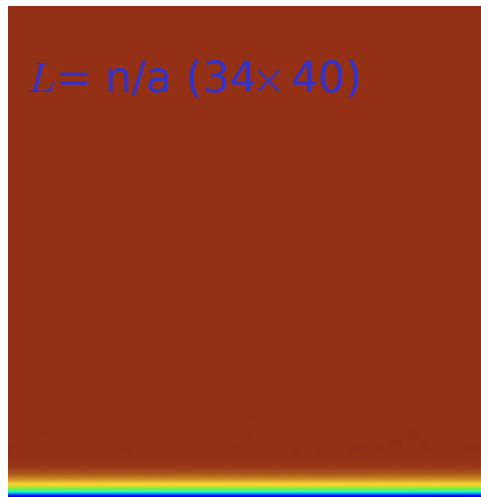
Relaxation Term Necessary drain of energy (SGS dissipation), acting on the small scales (large wavenumber) only

Reference: Schlatter *et al*, *Int. J. Heat Fluid Flow* (2004)

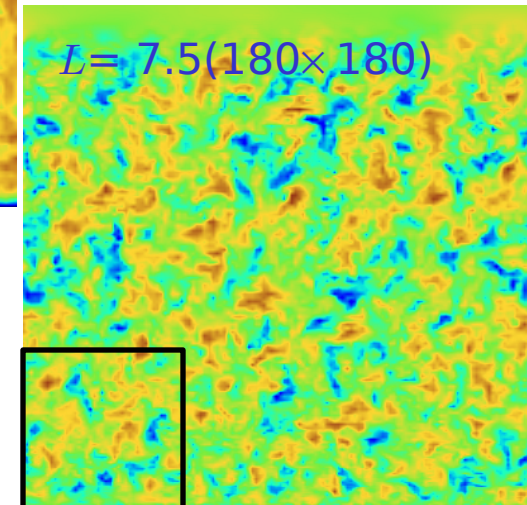
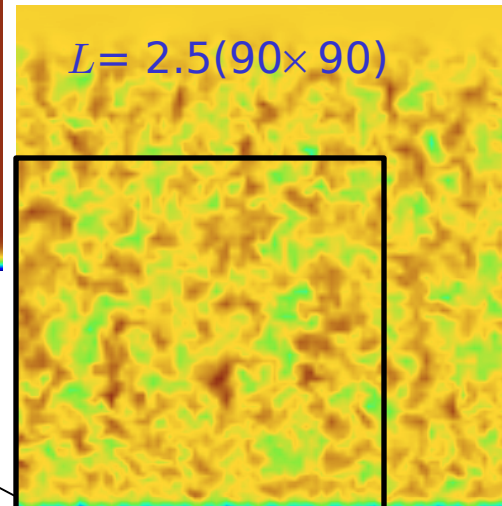


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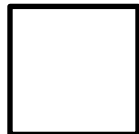
Free-Stream Turbulence



Streamwise velocity in the Y-Z plane at the inlet for different length scale



L increases



Brandt, *et al.*, *JFM*(2004), $L=5$

Similar method is used by Jacobs & Durbin, *JFM* (2001)
The generated FST is homogeneous and isotropic

Main Parameters

	$x_l \times y_l \times z_l$ δ_0^*	$N_x \times N_y \times N_z$ (grid points)	$Re_{\delta_0^*}$
Box 1	$1500 \times 90 \times 90$	$384 \times 121 \times 96$	300
Box 2	$1500 \times 180 \times 180$	$384 \times 201 \times 128$	300
Box 3	$1500 \times 180 \times 240$	$384 \times 201 \times 192$	300
Box 4	$750 \times 40 \times 34$	$1024 \times 289 \times 128$	450

	Tu %	L δ_0^*	Box
Case 1	0	n/a	Box 1
Case 2	4.7, 20	2.5, 5, 7.5	Box 1
Case 3	4.7, 20, 40	5, 7.5, 15	Box 2
Case 4	4.7, 20, 40	7.5, 15	Box 3
Case 5	0	n/a	Box 4

The grid spacing is $\Delta x^+ = 60$, $\Delta z^+ = 16$
 2 points below $y^+ = 1$ and 8 points below $y^+ = 10$

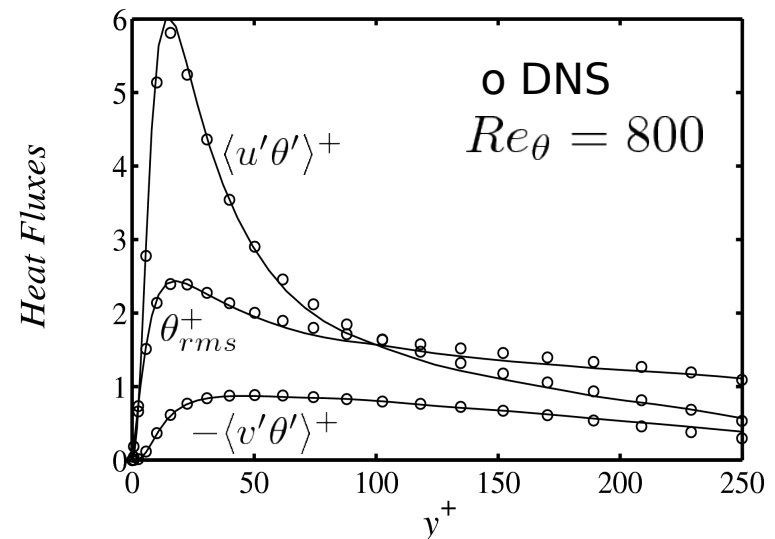
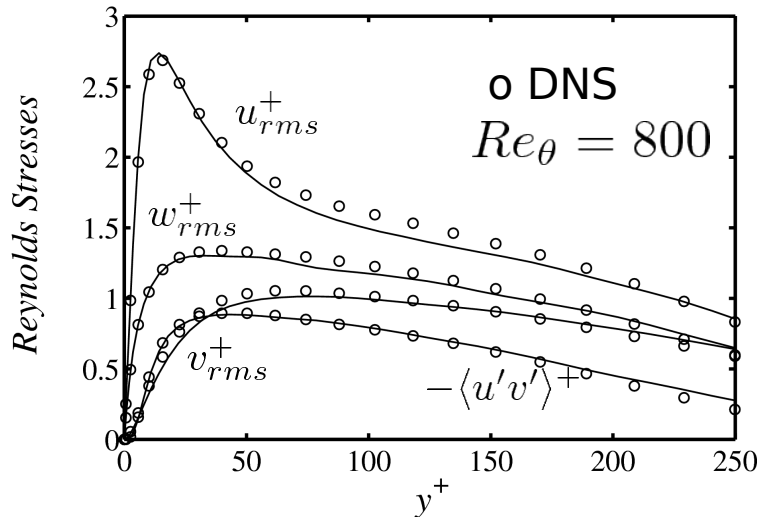
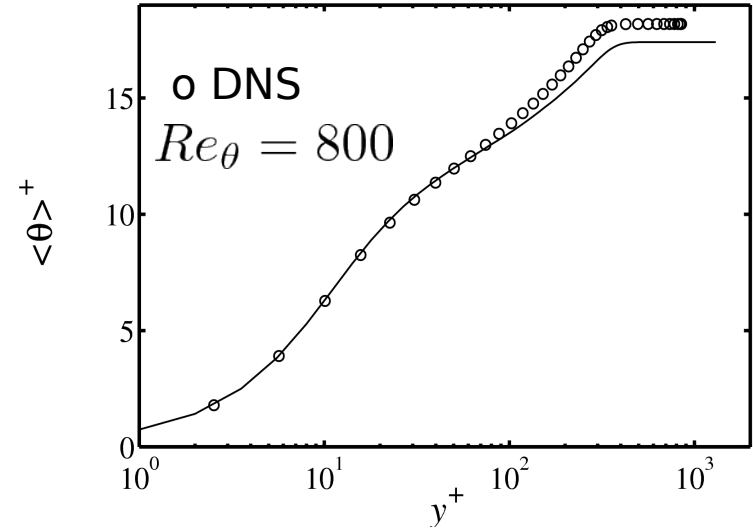
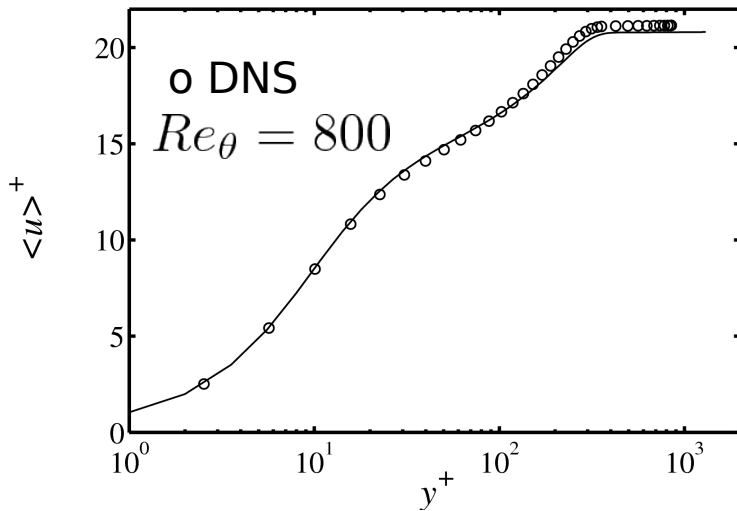


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Validation of the LES Results



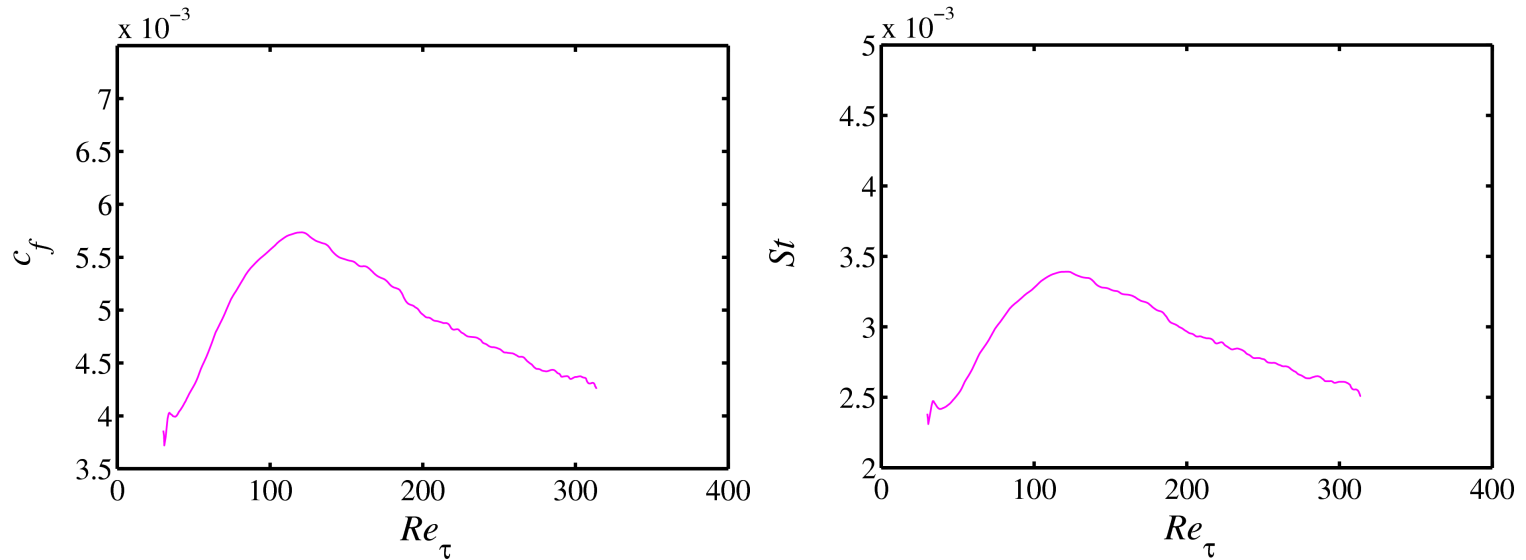
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DNS Reference: Li *et al*, (2009), *Int. J. Heat Fluid Flow*

Free-Stream Turbulence

Influence of the FST on c_f and St



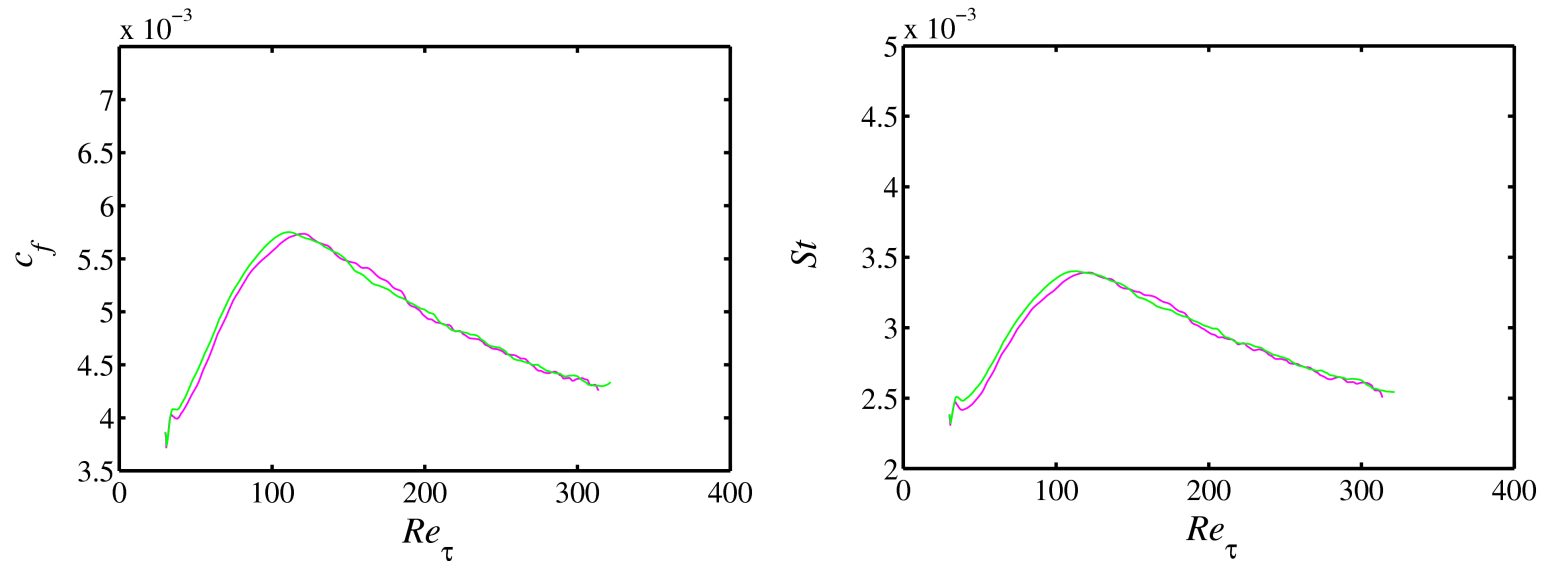
— $Tu = 0\%$, $L = 0$



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Free-Stream Turbulence

Influence of the FST on c_f and St



— $Tu=0\%$, $L=0$
 — $Tu=4.7\%$, $L=2.5$

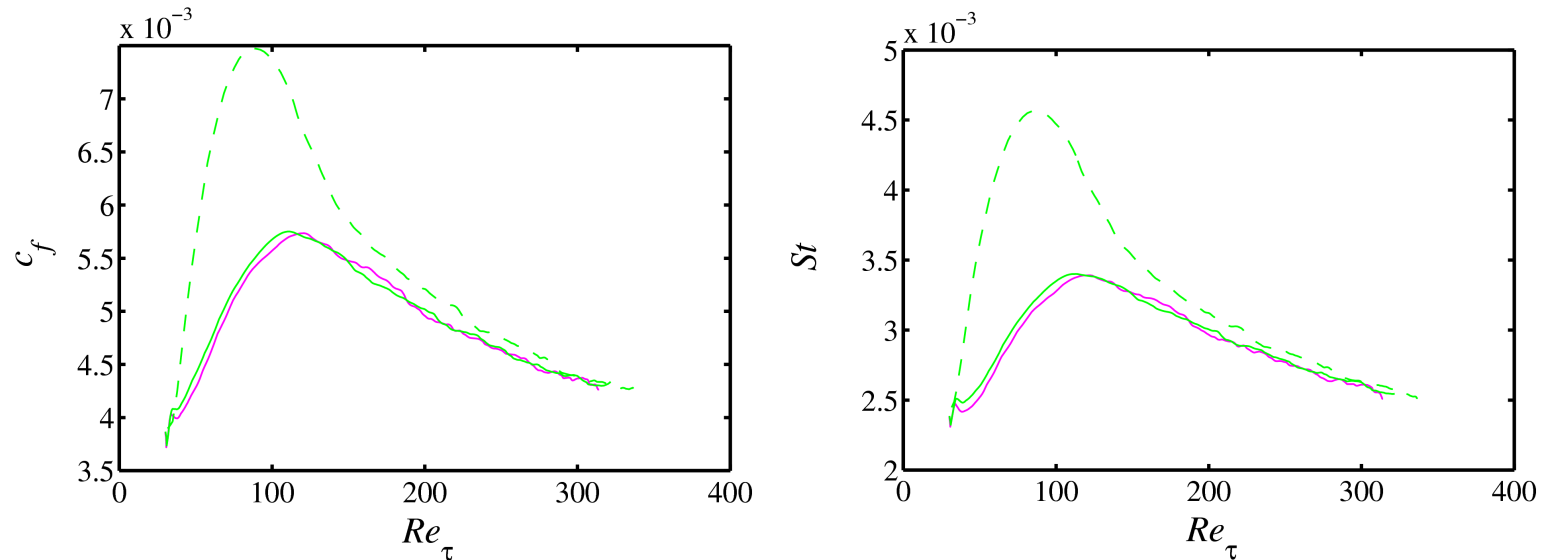
Due to a low $L \rightarrow$ similar to the case with no FST



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Free-Stream Turbulence

Influence of the FST on c_f and St



- $Tu= 0\%, L=0$
- $Tu=4.7\%, L=2.5$
- - $Tu= 20\%, L=2.5$

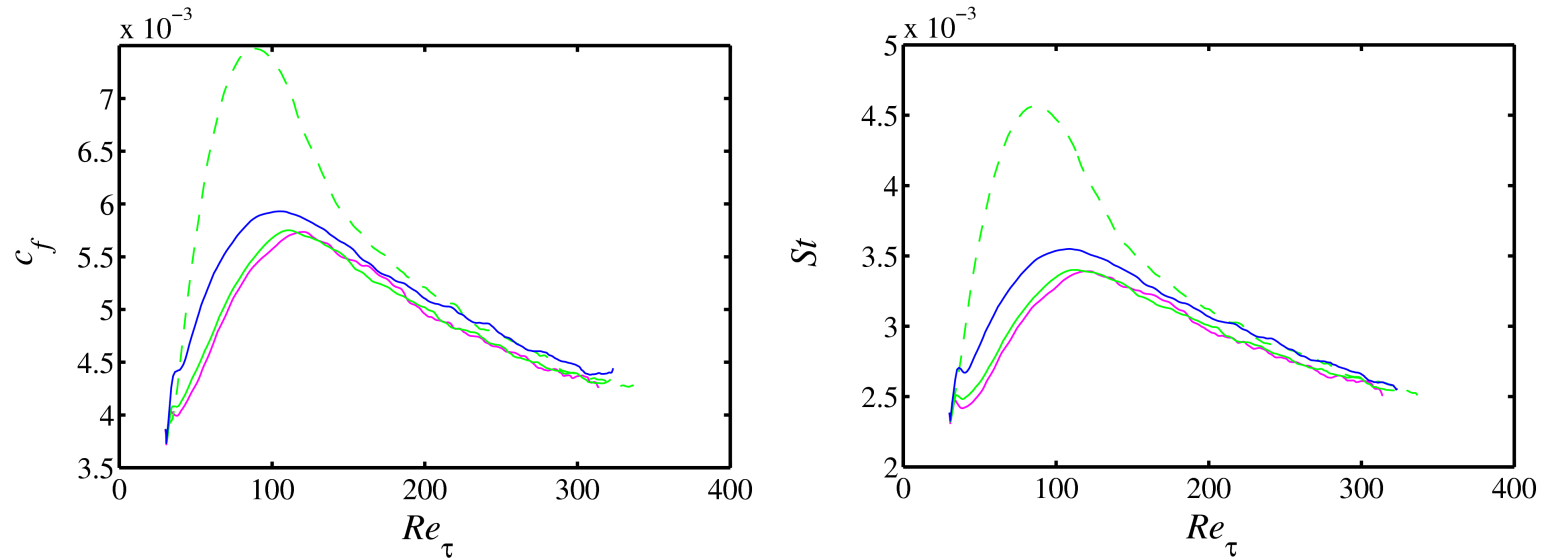
Due to a low $L \rightarrow$ similar to the case with no FST



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Free-Stream Turbulence

Influence of the FST on c_f and St



- $Tu=0\%$, $L=0$
- $Tu=4.7\%$, $L=2.5$
- - - $Tu=20\%$, $L=2.5$
- $Tu=4.7\%$, $L=7.5$

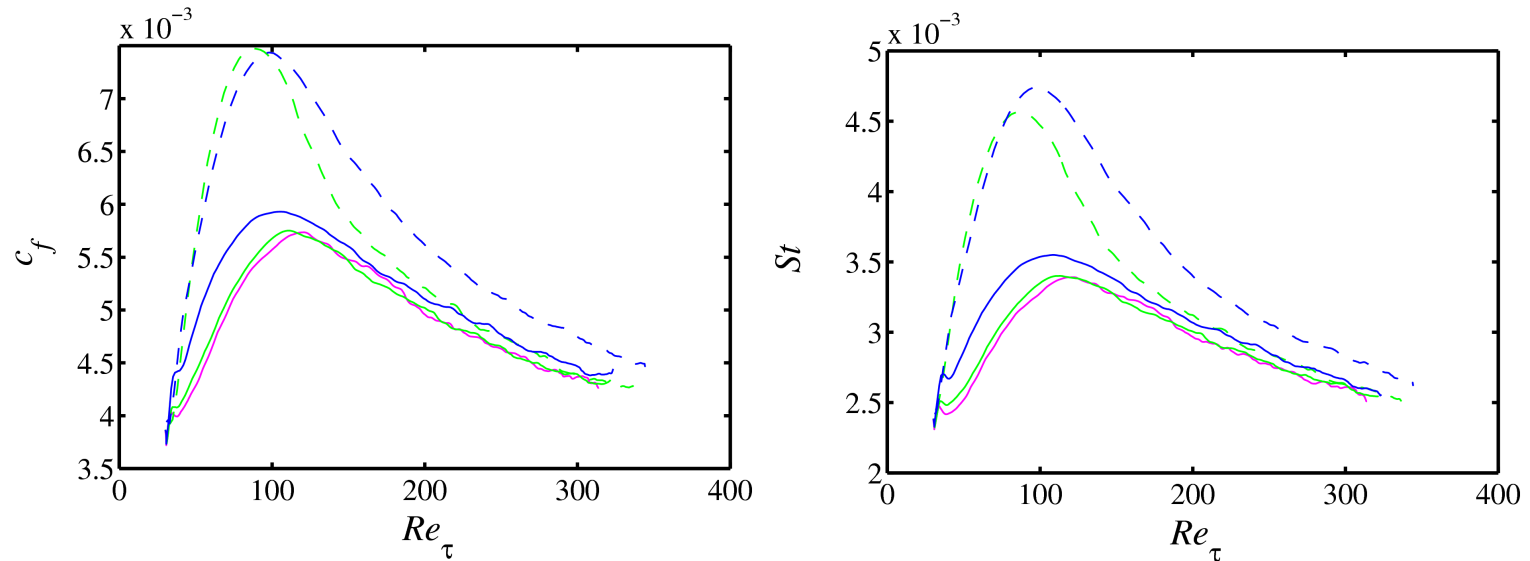
Even L is bigger, due to the low Tu , effects are not obvious



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Free-Stream Turbulence

Influence of the FST on c_f and St



- $Tu = 0\%$, $L = 0$
- $Tu = 4.7\%$, $L = 2.5$
- - $Tu = 20\%$, $L = 2.5$
- $Tu = 4.7\%$, $L = 7.5$
- - $Tu = 20\%$, $L = 7.5$

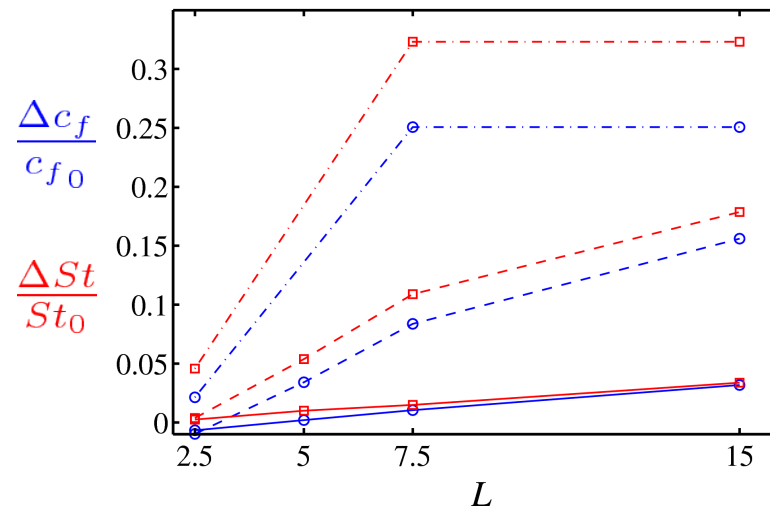
Both Tu and L are important to maintain the effects of FST



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Free-Stream Turbulence

Relative increase of c_f and St at $Re_\tau = 300$



— inlet $Tu = 4.7\%$, - - - inlet $Tu = 20\%$, - · - inlet $Tu = 40\%$

For smaller L , St increase more than c_f , but not for larger L

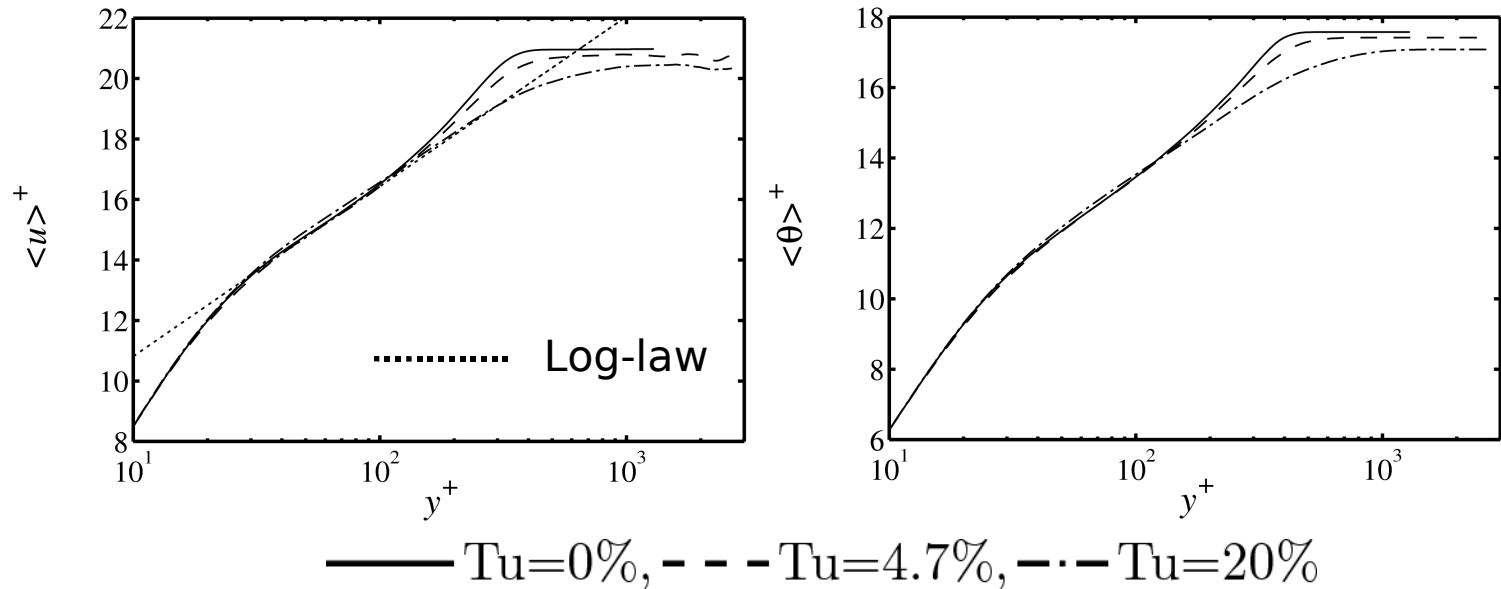
For $L=7.5$, local $Tu=7\%$ → a relative increase of c_f **32%** and St **25%**



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Free-Stream Turbulence

Influence of the FST on the mean profiles at $Re_\theta = 850$



Depression of the boundary-layer wake region starts appearing with increasing Tu

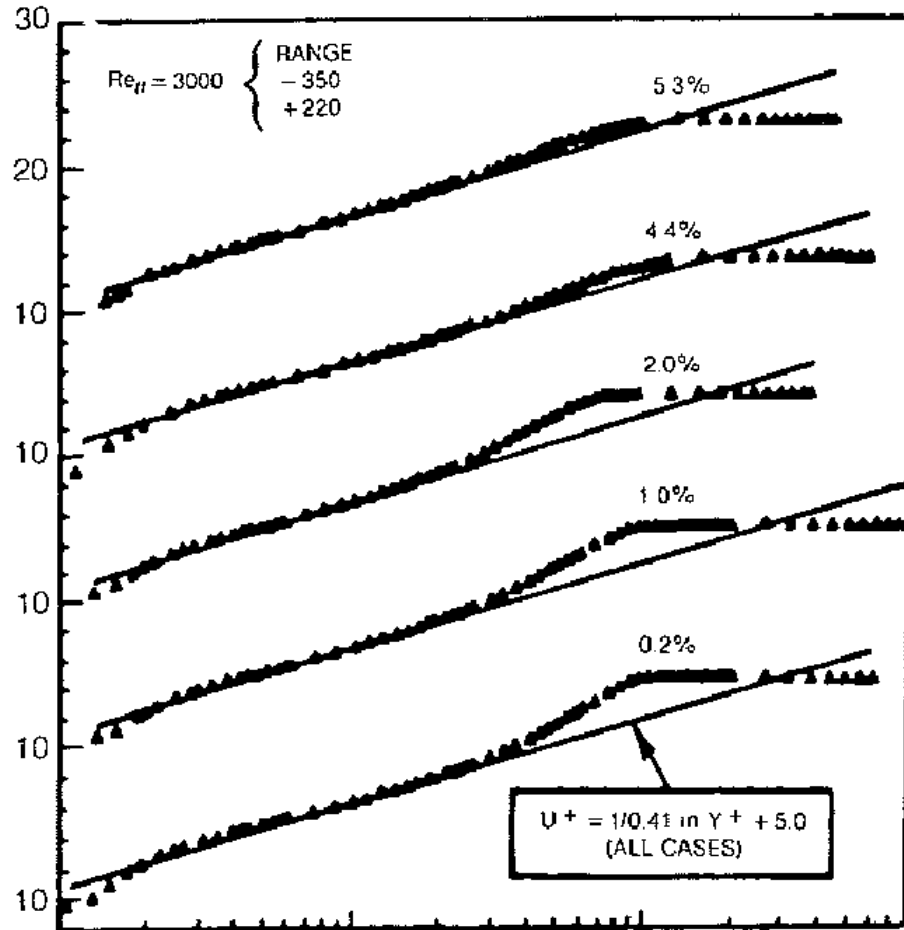
Viscous (conductive) sub-layer and logarithmic region are insensitive to the FST



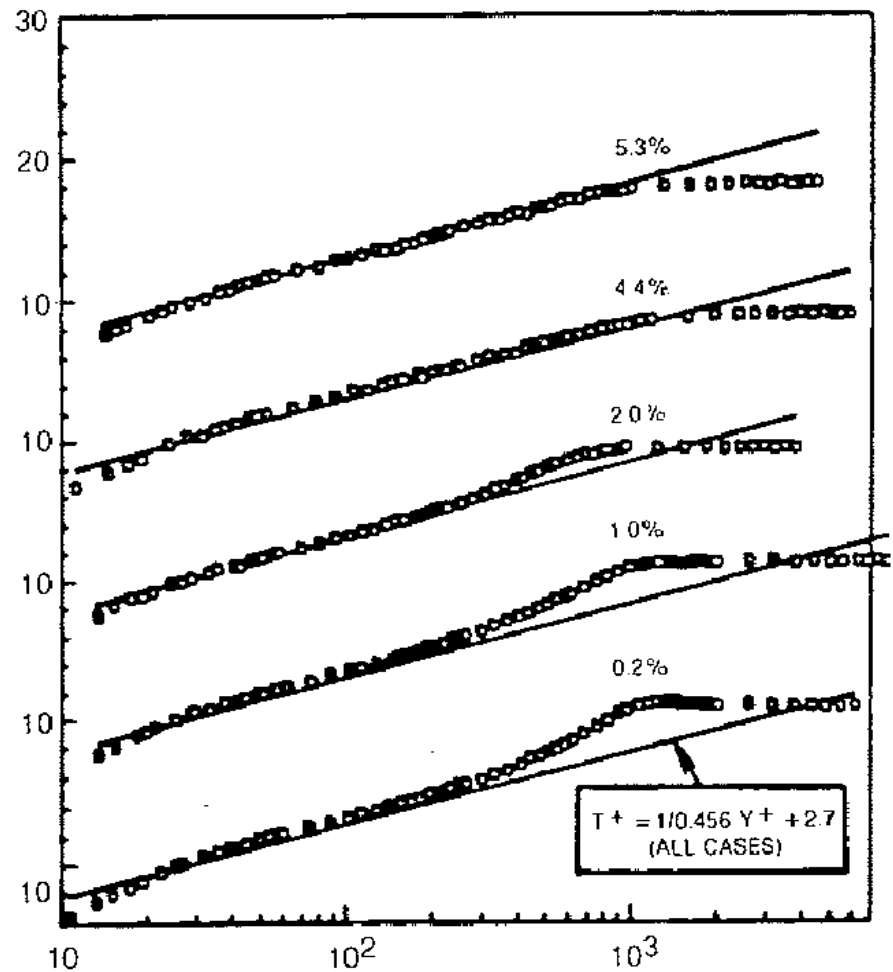
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Free-Stream Turbulence

Mean velocity profile



Mean temperature profile

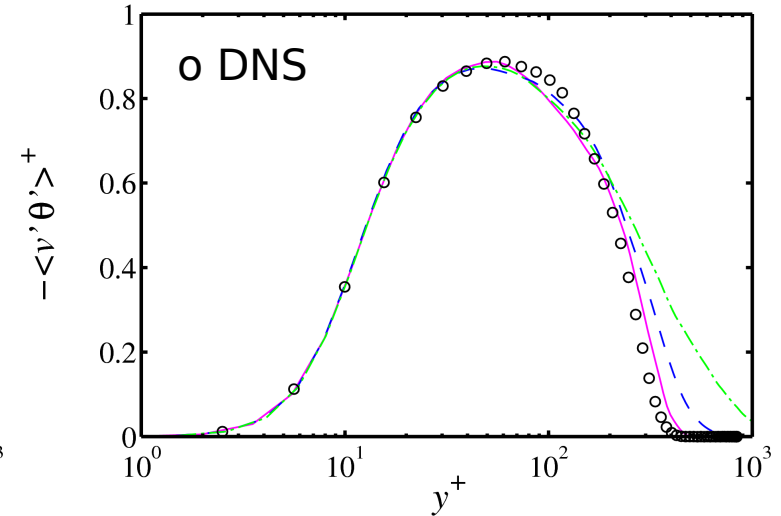
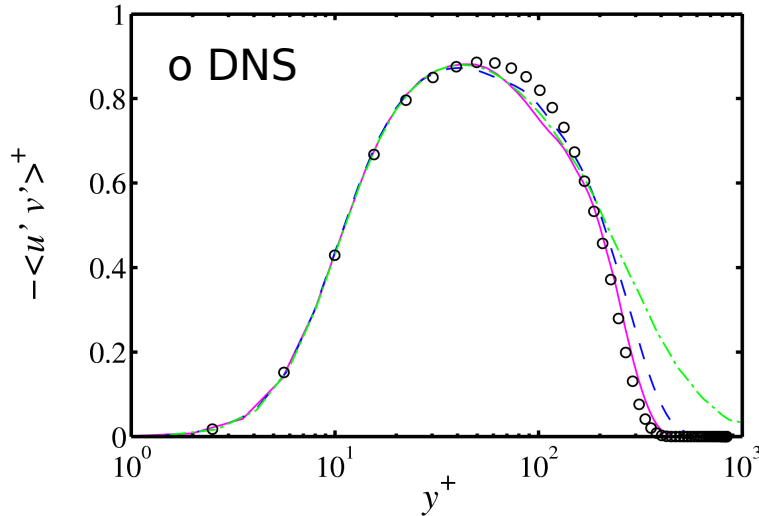


Blair, 1983

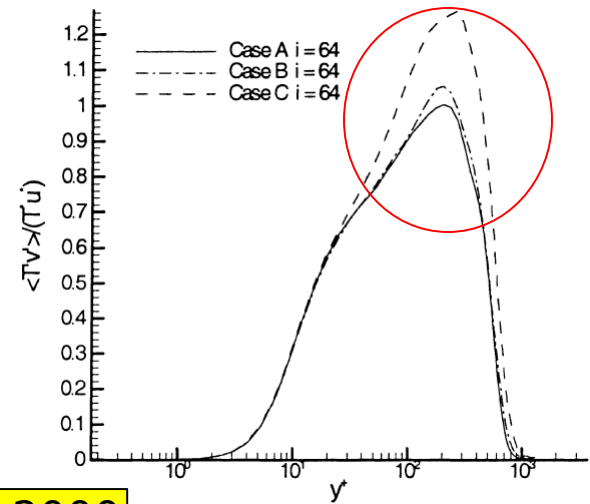
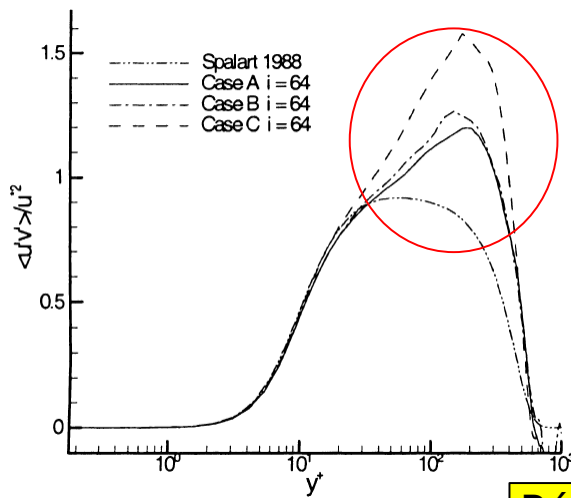
Free-Stream Turbulence



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— $Tu = 0\%$, - - - $Tu = 4.7\%$, - · - · $Tu = 20\%$



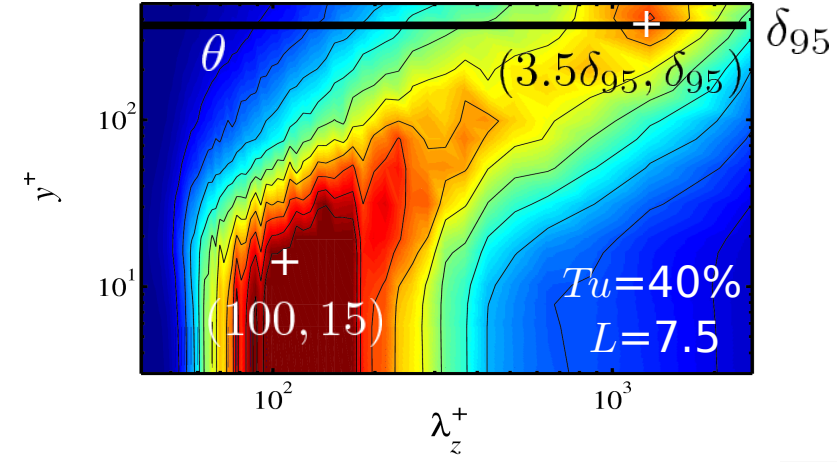
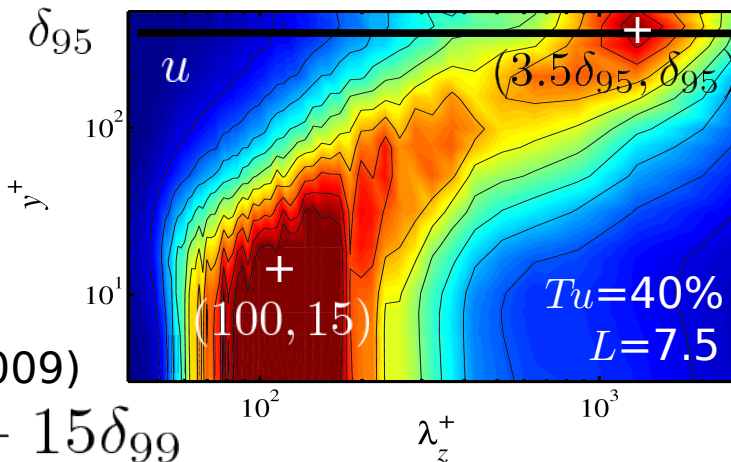
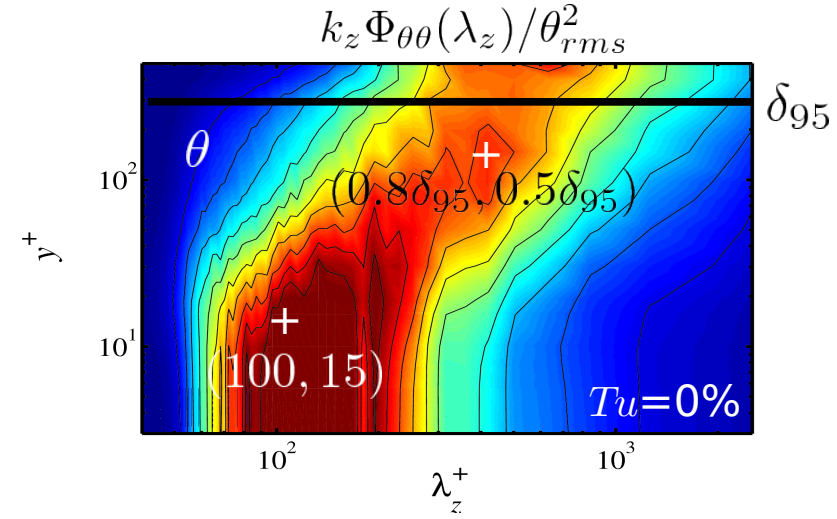
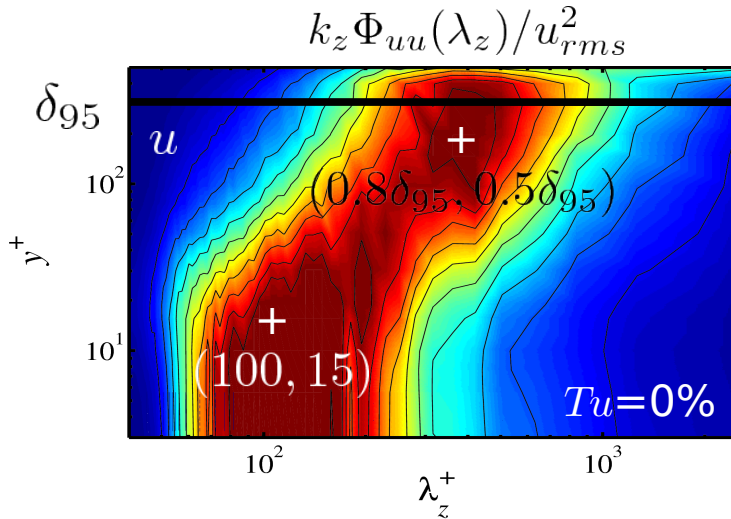
Péneau et al., 2000

Free-Stream Turbulence

Premultiplied spanwise energy spectra of u and θ at $Re_\theta = 900$



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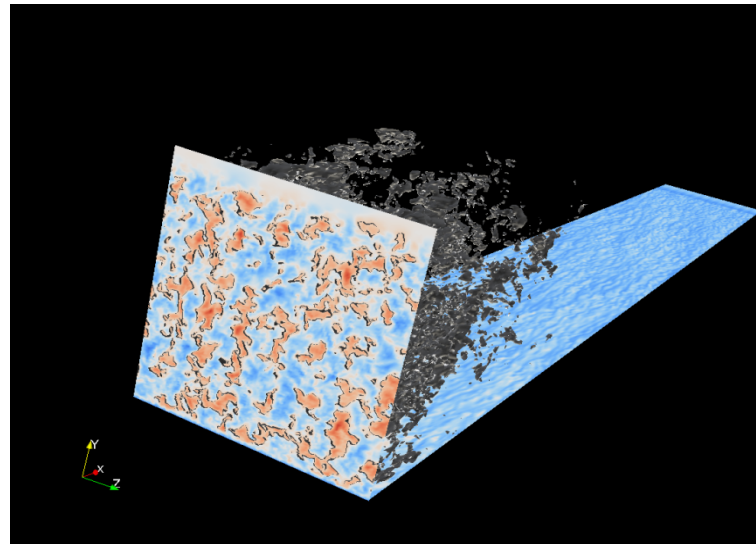


Sharp et al. (2009)
 $\lambda_x \sim 6\delta_{99} - 15\delta_{99}$

Conclusions & Outlook



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- A series of LES of TBL with FST has been performed, the newly implemented LES models give good results at a low cost (5%-10%)
- Increased c_f and St with increasing Tu are observed, for small L , St grows faster than $c_f \rightarrow$ Reynolds analogy factor **grows** with Tu
- The increase is due to footprints of the **large-scale motions** near the boundary layer edge
- Depression of the wake region is associated with the **lower intermittency** due to FST
- DNS will be performed, more detailed analysis will be done, in particular compare to results without FST at high Reynolds number

Comparison of TBL and Channel for Heat Transfer



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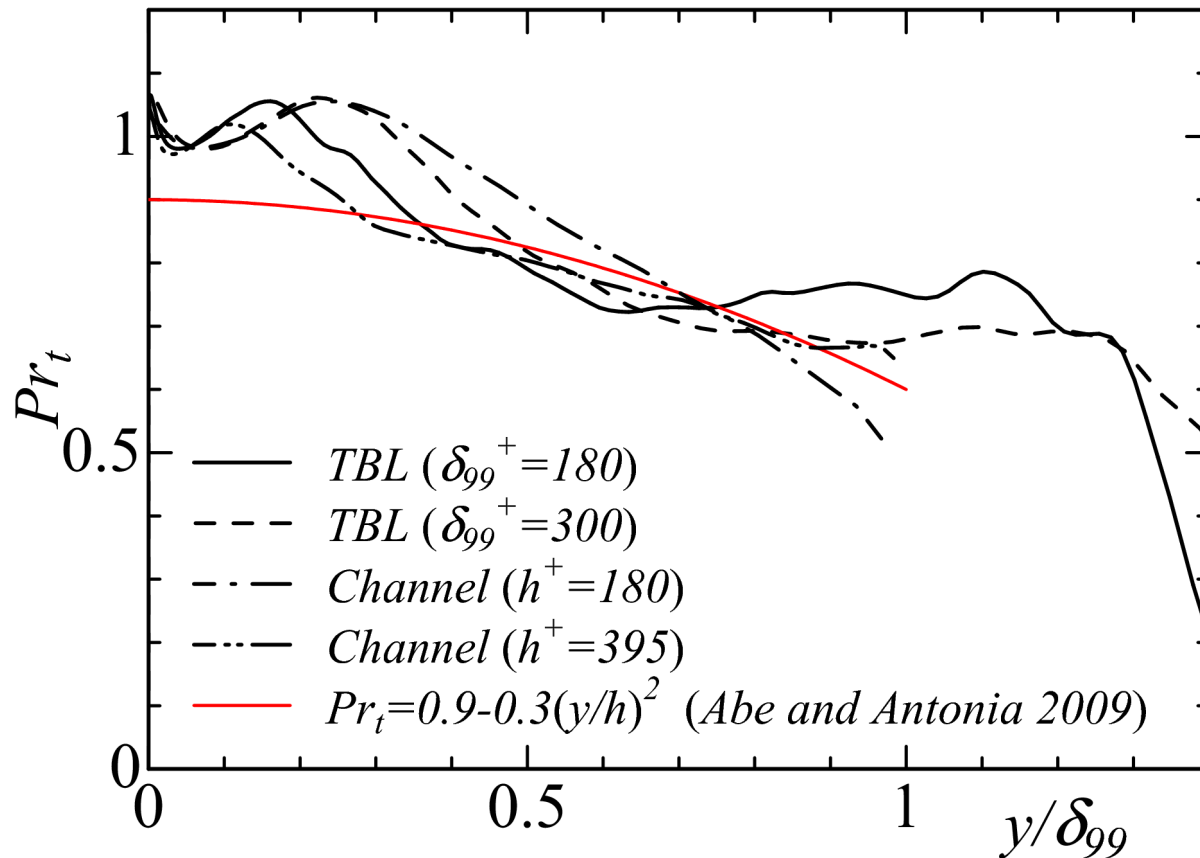
FLOW

Preliminary result

Turbulent Prandtl Number

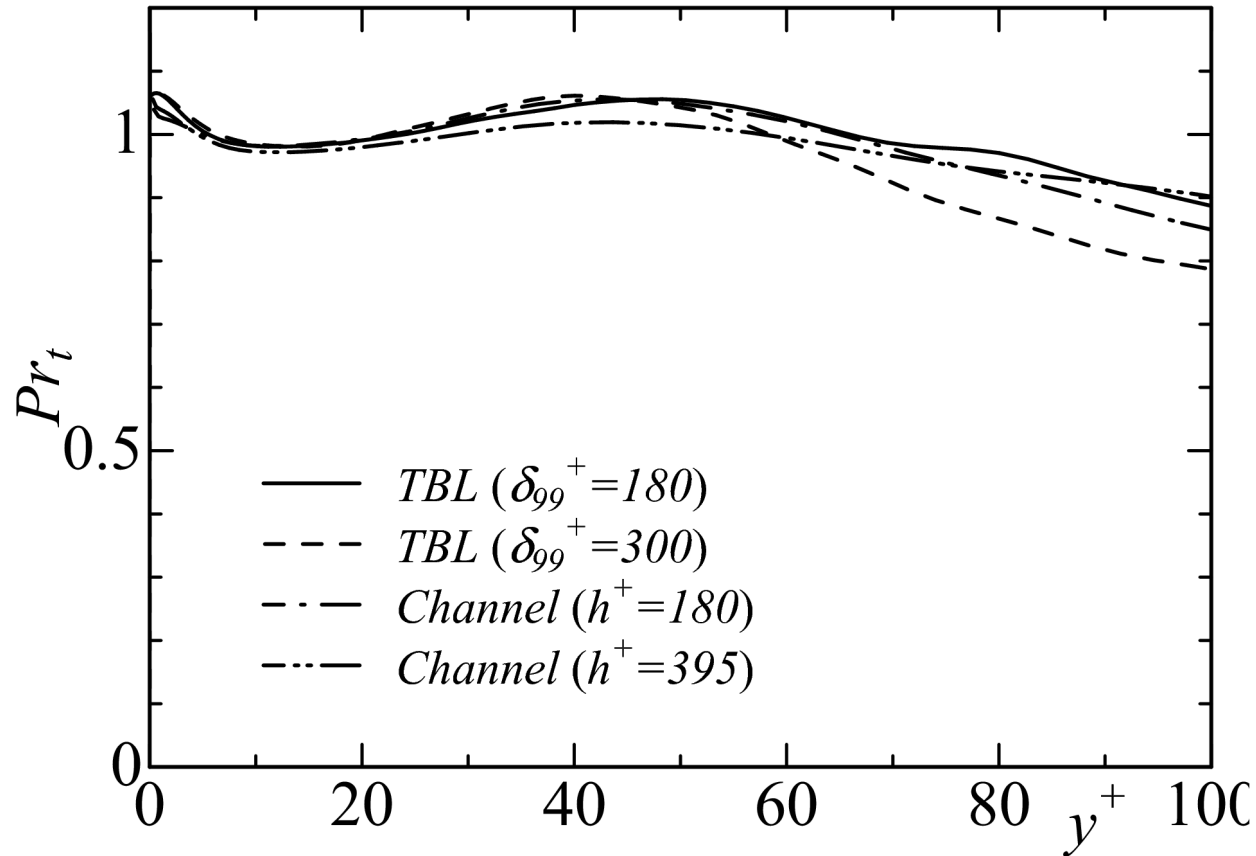
$$\nu_t = -\frac{\langle u'v' \rangle}{\frac{\partial \langle u \rangle}{\partial y}} \quad \text{Turbulent eddy viscosity}$$

$$\alpha_t = -\frac{\langle v'\theta' \rangle}{\frac{\partial \langle \theta \rangle}{\partial y}} \quad \text{Turbulent eddy diffusivity}$$



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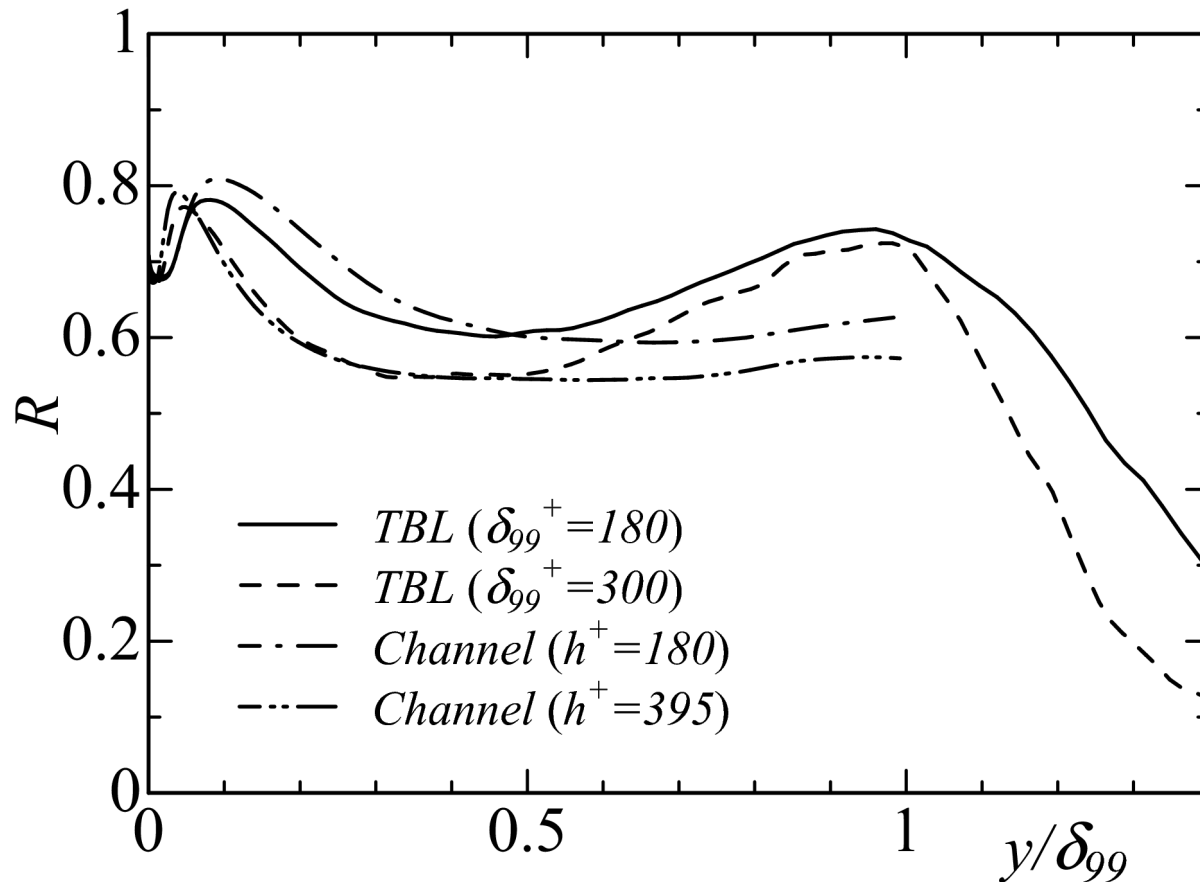
Turbulent Prandtl Number



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Time Scale Ratio

$$R = \frac{k_{\theta}/\varepsilon_{\theta}}{k/\varepsilon}$$



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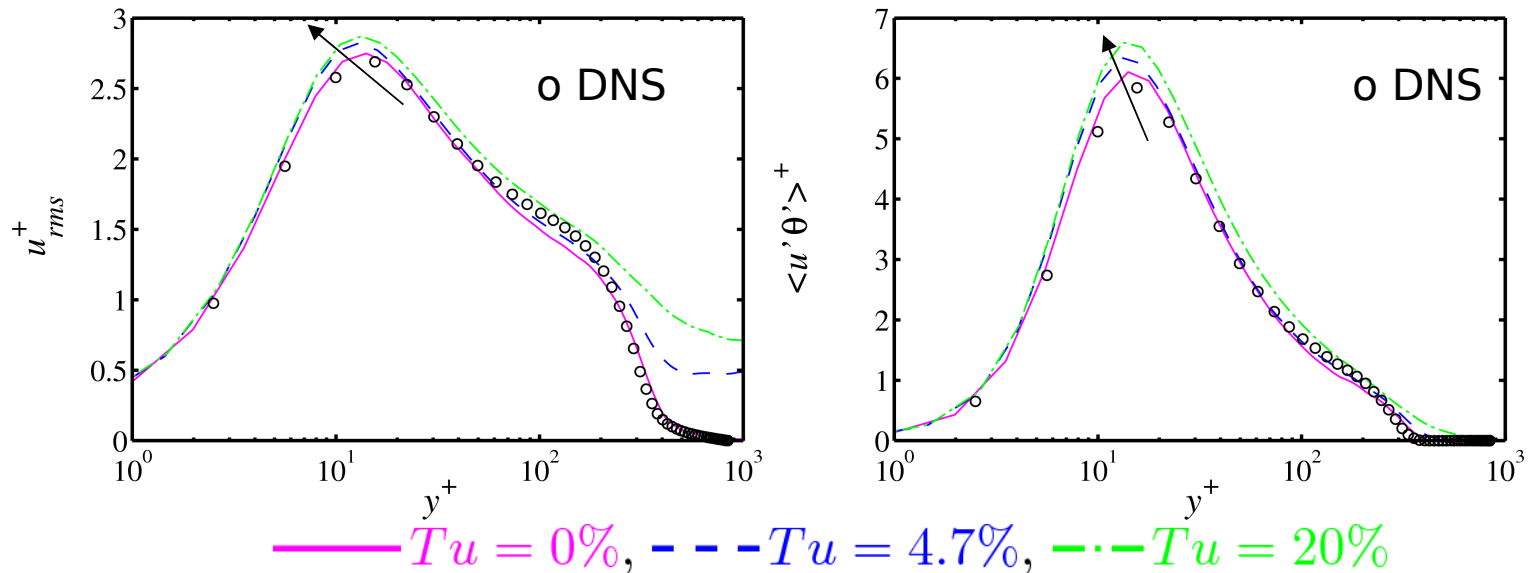


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Thank you!

Free-Stream Turbulence

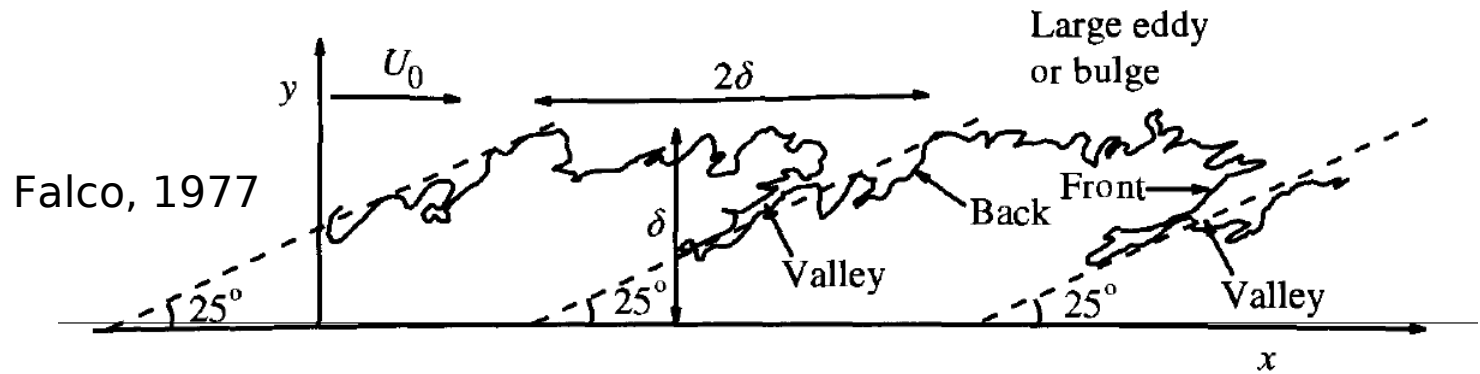
Influence of the FST on the Reynolds stress heat flux at $Re_\theta = 850$



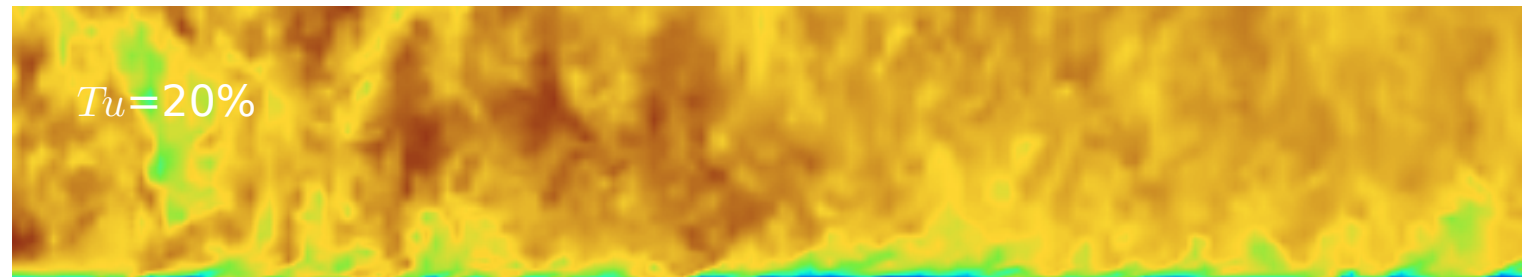
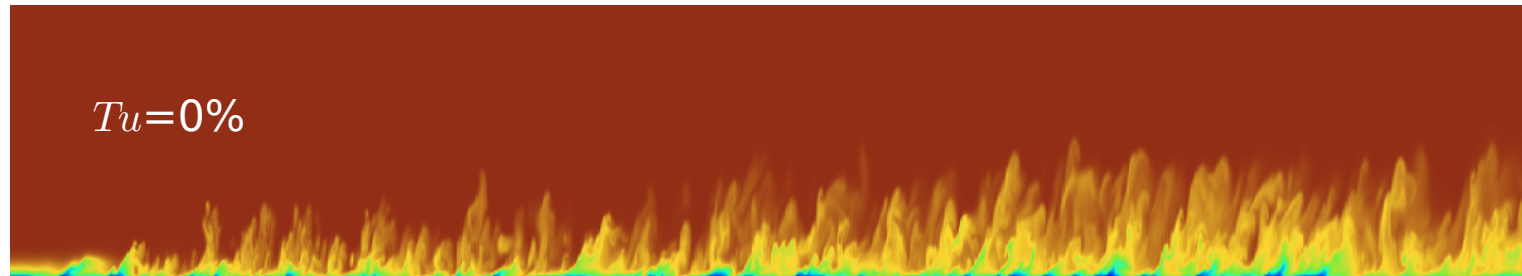
Peak moves towards the wall and increases with higher Tu

More increases in the free stream

Free-Stream Turbulence



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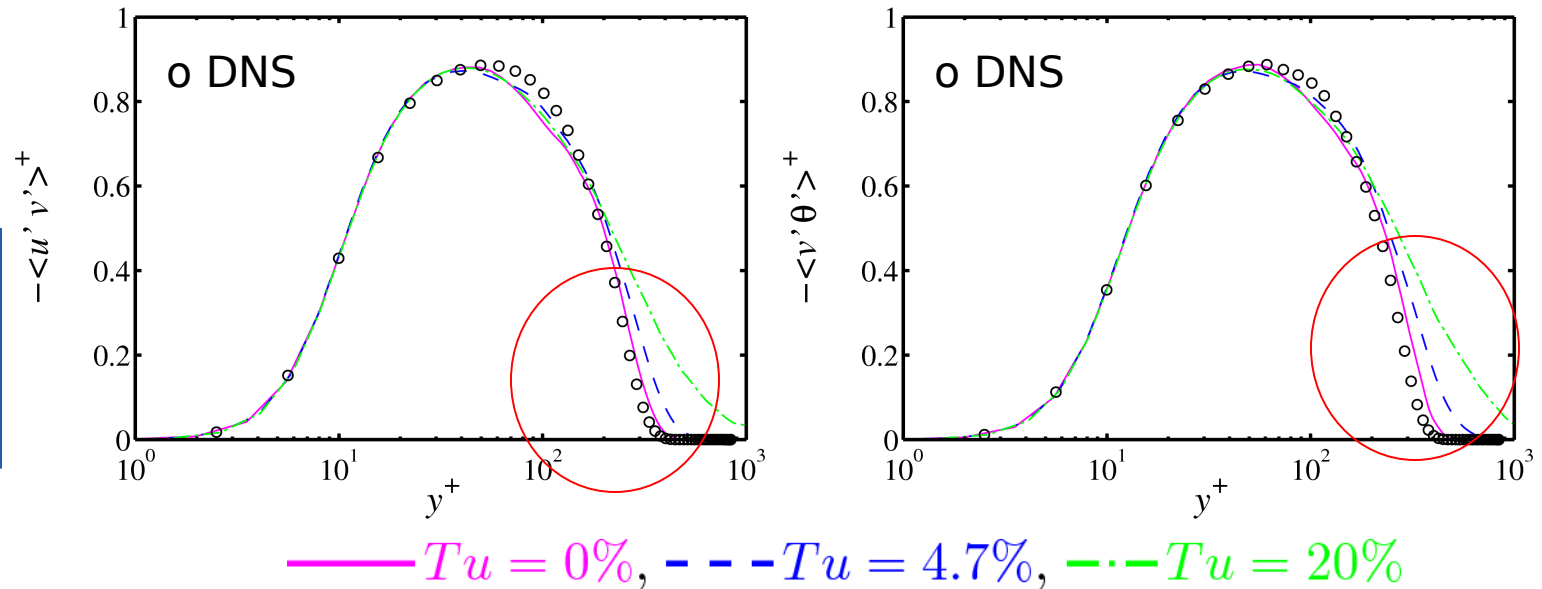


Depression of wake region is due to the loss of intermittency

Free-Stream Turbulence



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The increase of c_f and St is connected to the large-scale structure resides in the outer region